

Q. $h[n] = 5^n u[n]$

Ans. option (d) is correct.

Q. if $H(z) = \frac{1}{(1 + \frac{1}{2}z^{-1})(1 - \frac{1}{4}z^{-1})}$

ES-03
15 marks

Represent $H(z)$ in

1. Direct form
2. Parallel "
3. Cascade "

Ans. 1) Direct form:-

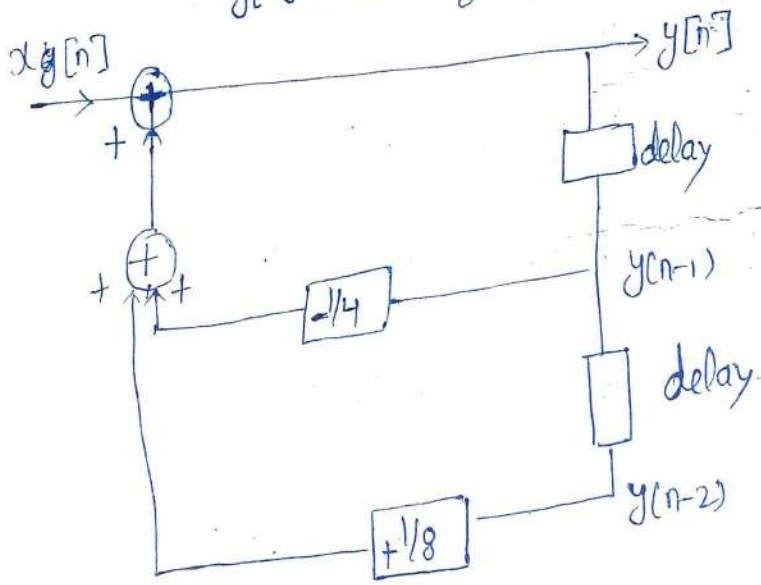
$$\frac{Y(z)}{X(z)} = \frac{1}{1 - \frac{1}{4}z^{-1} + \frac{1}{2}z^{-1} - \frac{1}{8}z^{-2}}$$

$$\frac{Y(z)}{X(z)} = \frac{1}{1 + \frac{1}{4}z^{-1} - \frac{1}{8}z^{-2}}$$

$$Y(z) + \frac{1}{4}z^{-1}Y(z) - \frac{1}{8}z^{-2}Y(z) = X(z)$$

$$y[n] + \frac{1}{4}y[n-1] - \frac{1}{8}y[n-2] = x[n]$$

$$y[n] = x[n] + \frac{1}{8}y[n-2] - \frac{1}{4}y[n-1]$$



2) Parallel form:-

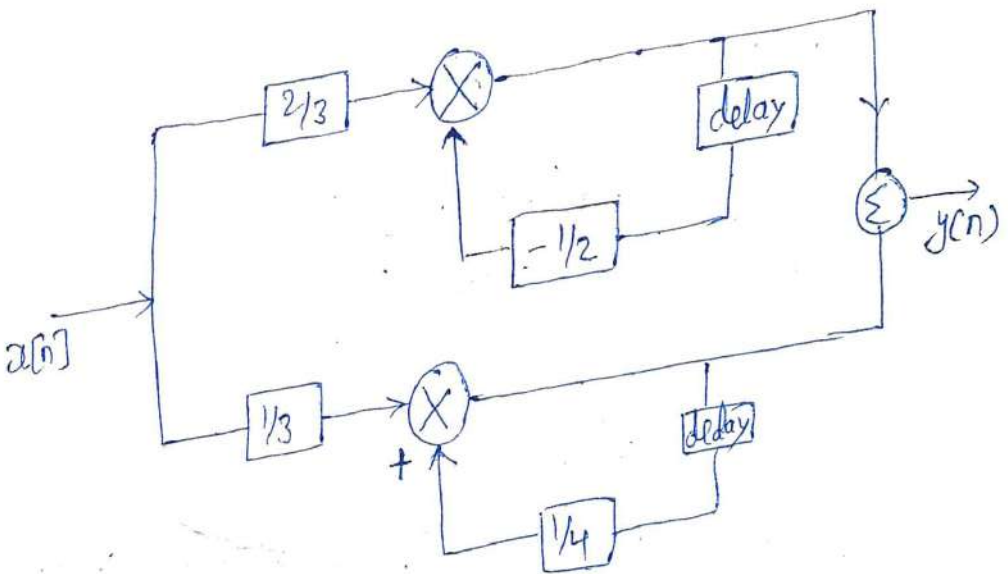
$$H(z) = \frac{1}{(1 + \frac{1}{2}z^{-1})(1 - \frac{1}{4}z^{-1})}$$

Partial fraction

$$\frac{Y(z)}{X(z)} = \frac{2/3}{1 + \frac{1}{2}z^{-1}} + \frac{1/3}{1 - \frac{1}{4}z^{-1}}$$

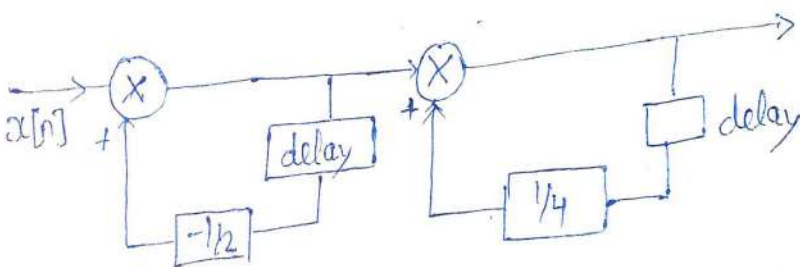
$$H(z) = H_1(z) + H_2(z)$$

$$H_1(z) = \frac{Y_1(z)}{X_1(z)} = \frac{2/3}{1 + \frac{1}{2}z^{-1}}$$



3) Cascade form →

$$H(z) = \frac{1}{1 + \frac{1}{2}z^{-1}} \times \frac{1}{1 - \frac{1}{4}z^{-1}}$$



Q. 98 $H(z) = \frac{z^2 + 1}{(z - 0.5)(z + 0.5)}$

What is the value of $H(1)$?

Ans.

$$h(n) = \lim_{z \rightarrow \infty} \frac{dt}{z} \frac{1 + z^{-2}}{(1 - 0.5z^{-1})(1 + 0.5z^{-1})}$$

$$= 1$$

As poles lie in unit circle so it is stable system.

Q. $x[n] = (1/4)^n u[-n] = (4)^{-n} u[-n]$

Then what is Z T F?

A) $\frac{4z}{z-1}$, $|z| > 1/4$ c) $\frac{1}{1-4z}$ $|z| > 1/4$

B) $\frac{4z}{4z-1}$ $|z| < 1/4$ d) $\frac{1}{1-4z}$ $|z| < 1/4$ (✓)

Ans.

$$4^n u[n] = \frac{1}{1-4z^{-1}}$$

$$4^{-n} u[-n] = \frac{1}{1-4z}$$

Q. $x[n] = (2/3)^{|n|}$

A. $\frac{-5z}{(z^2-3)(3z-2)}$ $-3/2 < |z| < -2/3$ (C) $\frac{5z}{(z^2-3)(3z-2)}$ $2/3 < |z| < 3/2$ (✓)

B. $\frac{-5z}{(z^2-3)(3z-2)}$ $2/3 < |z| < 3/2$ (d) $\frac{5z}{(z^2-3)(3z-2)}$ $3/2 < |z| < 2/3$

Ans.

$$x[n] = b^{|n|}$$

ROC is

$$b < |z| < 1/b \text{ where } 0 < b < 1.$$

$$z = \frac{1 \pm \sqrt{1 + \frac{1}{2}}}{2} = \frac{1 \pm \sqrt{2}}{2}$$

$$z_1 = 1.2, -0.2$$

↳ lie outside of unit circle so unstable.

Note: -

1. for stable system all pole must lie inside unit circle
2. for min. phase all poles & zeros must lie inside the unit circle.

Q. 9 f $H(z) = \frac{4z^{-1}}{(1 - \frac{1}{4}z^{-1})^2}$, $|z| > \frac{1}{4}$

then system is

1. Causal

2. Stable

A) 1

B) only stable (2)

C) 1/2 (✓)

D) None

Ans.

$$h[n] = 4 \times (1/4)^n u[n]$$

$$= 4 \times (1/4)^{n-1} u[n-1]$$

So, causal system, stable

Q. $H(z) = \frac{\frac{1}{4}z^{-1}}{(1 - \frac{1}{4}z^{-1})^2}$, $|z| > \frac{1}{4}$

Ans. $h[n] = (1/4)^n u[n]$, $(\frac{1}{4})^n u[n] \rightarrow \frac{1}{(1 - \frac{1}{4}z^{-1})}$

$$n (\frac{1}{4})^n u[n] \rightarrow -z \times \frac{d}{dz} \left(\frac{1}{1 - \frac{1}{4}z^{-1}} \right)$$

$$h[n] = n (1/4)^n u[n]$$

$n < 0, h[n] = 0$, system causal

Time Response of System:-

- 1) Zero-step → In this case g/p is zero & o/p is only due to initial condition
- 2) Zero- g/p → In this case initial cond's are zero & o/p is only due to g/p .

IES

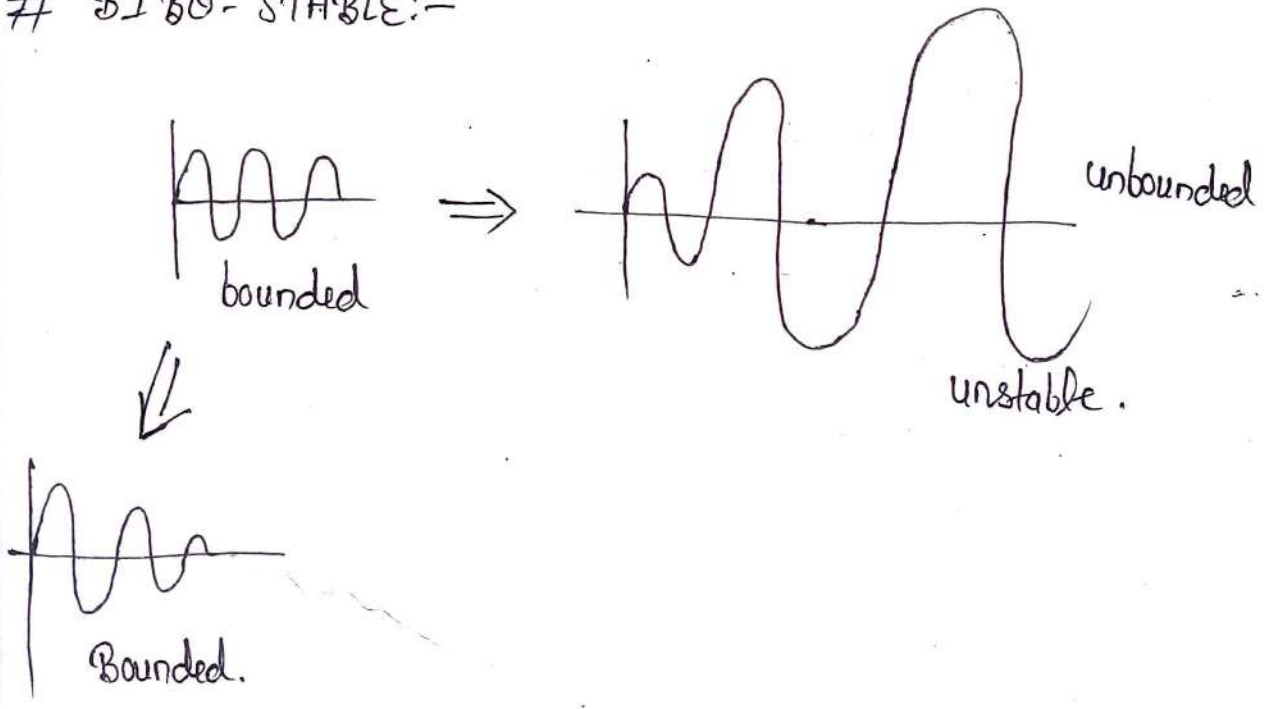
Q. Transfer function is defined for

- 1. Zero step
- 2. Zero g/p

- a) 1 (✓) c) 1/2
- b) 2 d) None

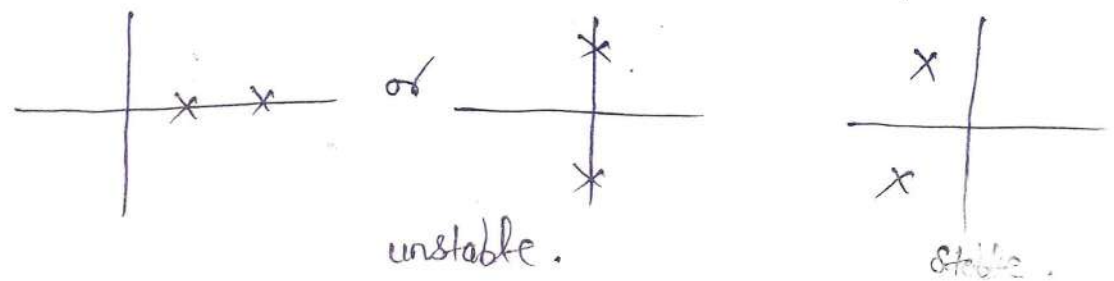
Total Response = Zero step + Zero g/p .

BIBO-STABLE:-



condition for BIBO-stable →

- (i) Roots must be on left hand side i.e., -ve real axis.
- (ii) If roots are on +ve real axis or on $j\omega$ -axis then it is not stable system.

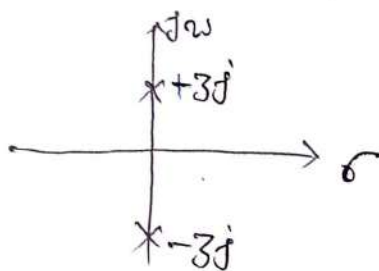


Asymptotic stable or Zero slip stable \rightarrow

condition \rightarrow 1. $|y(t)| < \infty$
2. $\lim_{t \rightarrow \infty} |y(t)| = 0$ } con. for asymptotic stable

Location of roots must be only on -ve real axis

If roots are on jw-axis then system is called as Marginally stable or Marginally unstable system. i.e. oscillation are present.



Q. $M(s) = \frac{20}{(s+1)(s+2)(s+3)}$ } Stable system

as roots are on -ve real axis.

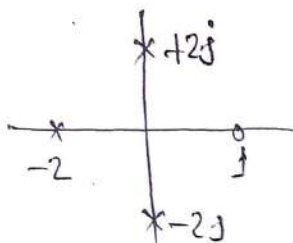
Q. $M(s) = \frac{20(s-1)}{(s+2)(s^2+4)}$

a) stable

c) Marginally stable

b) unstable

d) Marginally stable or unstable (✓)



Q. $M(s) = \frac{20(s+1)}{(s-1)(s^2+2s+2)}$

unstable system due to poles on -ve real axis.

Q. $M(s) = \frac{20}{(s^2+4)(s+10)}$ (2)

Marginally stable or unstable.

If multiple odd order roots are present on jw-axis then system will become unstable.

Q. $M(s) = \frac{20}{(s^2+4)^2(s+10)}$

unstable

Q. $M(s) = \frac{20}{(s^2+2s)}$ or $\frac{20}{s(s+2)}$

If any pole is placed intentionally then system is stable.

Q. $M(s) = \frac{10}{s^4+3as^3+s^2+10s}$



Stable system.

Transient Response & Steady state Response →

$y(t) = 1 + e^{-4t}$
 → Transient Response
 → Steady-state

Transient Response:- It is defined as part of response that goes to zero as time become very large so $\lim_{t \rightarrow \infty} y(t) = 0$

this is condition for transient response.

Steady-state:- It is part of total response that remains after transient has died out.

If $\lim_{t \rightarrow \infty} y(t) \neq 0$ is cond. for steady state.

Q. When time period of observation is large then type of error is

Soln

a) Transient Error

c) Half-power Error

b) Steady state Error (✓)

d) Position Error const.

Q. Which response remain nature of response (oscill. or overdamp)

IES

a) Transient (✓)

c) Half-power

b) Steady state

d) Position

So, Steady-state talk part remains accuracy of C.S.

Transient

Steady-state

i) Time Response upto

3T or 4T

ie) After 4T is called as

Steady state

ii) It doesn't depend on input signal

ii) It depend on input signal.

iii) It reveals nature of response & given indication about speed.

iii) It reveals accuracy.

iv) Natural response

iv) forced response.

Q. In case initial condition of a system are specified to be zero, it implies that system is

A) working with zero ref. input

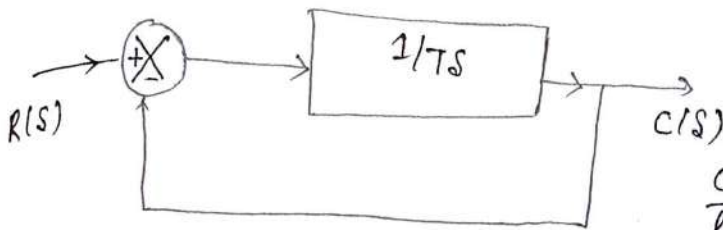
B) works with ref. input

C) at rest but stores energy

D) at rest without stored energy. (✓)

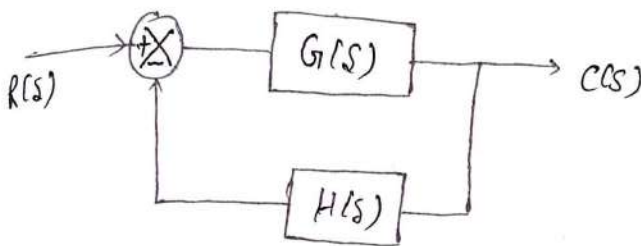
Time-Response of 1st order system:-

(3)



$$\frac{C(s)}{R(s)} = \frac{1/Ts}{1 + 1/Ts}$$

$$= \frac{1}{1 + Ts}$$



$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

Case I:- If ~~the~~ ~~input~~ ~~is~~ given as step fn.

$$r(t) = u(t)$$

$$\Rightarrow R(s) = 1/s$$

$$\frac{C(s)}{R(s)} = \frac{1}{1 + Ts}$$

$$C(s) = \frac{1}{s(1 + Ts)} = \frac{1}{s} - \frac{T}{1 + Ts}$$

or

$$[\because L e^{-at} = \frac{1}{s+a}]$$

$$= \frac{1}{s} - \frac{1}{s + 1/T}$$

$$C(s) = \frac{1}{s} - \frac{1}{s + \frac{1}{T}}$$

$$C(t) = u(t) - e^{-t/T} u(t)$$

$$C(t) = u(t) (1 - e^{-t/T})$$

Case 2:- If GIP is given as Impulse:- $\mathcal{L}\{t\} = \delta(t)$
 $R(s) = 1$

$$\frac{C(s)}{R(s)} = \frac{1}{1+Ts}$$

$$C(s) = \frac{1}{(1+Ts)} R(s)$$

$$= \frac{1}{T(1/T+s)} = \frac{1}{T} e^{-t/T}$$

Case 3 \rightarrow If GIP is Ramp $x(t) = t$

$$\mathcal{L}\{t^n\} = \frac{n!}{s^{n+1}}$$

$$R(s) = \frac{1}{s^2}$$

$$C(s) = \frac{1}{s^2(1+Ts)} = \frac{A}{1+Ts} + \frac{B}{s} + \frac{C}{s^2}$$

$$C(s) = \frac{1}{s^2} - \frac{1}{s} + \frac{1}{1+Ts}$$

$$= \frac{1}{s^2} - \frac{1}{s} + \frac{1}{(1/T+s)}$$

$$c(t) = t u(t) - T u(t) + T e^{-t/T} u(t)$$

GATE-98

Q. unit impulse response of a system having transfer fn.

$\frac{k}{s+a}$ as shown above what is value of k?

Solⁿ

$$O/P = (F.T.F) \cdot \mathcal{L}\{i/p\} \times \mathcal{L}\{j/p\}$$

$$C(s) = \frac{k}{s+a} \cdot s = \frac{k}{s+a}$$

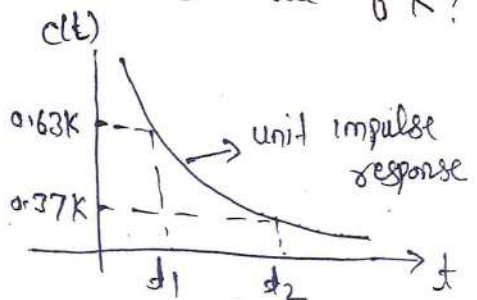
$$c(t) = k e^{-at}$$

$$= k e^{-(t/t_1)}$$

$$= ??$$

$$k e^{-(t/t_2)}$$

$$k e^{-1} = 0.37k$$

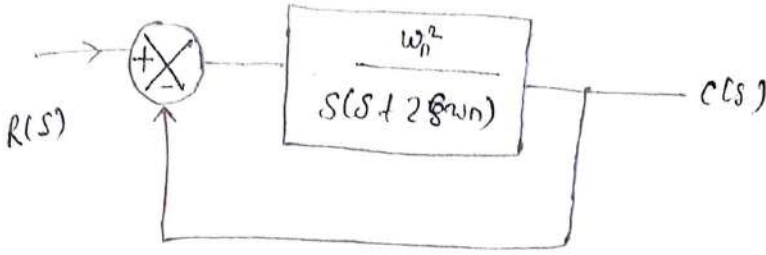


A) t_1 B) t_2

C) $1/t_1$ D) $1/t_2$ (✓)

IInd order system \rightarrow

(4)



$$\frac{C(s)}{R(s)} = \frac{\omega_n^2}{s(s + 2\zeta\omega_n)} \cdot \frac{1}{1 + \frac{\omega_n^2}{s(s + 2\zeta\omega_n)}} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = \frac{G(s)}{1 + G(s)H(s)}$$

Note
So, for calculating values of ζ & ω_n always use characteristic Eqn.

$$1 + G(s)H(s) = 0$$

Q. If $G(s)H(s) = \frac{225}{s^2 + 30s + 225}$ what is value of ζ & ω_n ?

$$\frac{225}{s^2 + 30s + 225} = \frac{G(s)}{1 + G(s)H(s)}$$

$$s^2 + 2\zeta\omega_n s + \omega_n^2$$

$$\omega_n^2 = 225 \Rightarrow \omega_n = 15$$

$$2\zeta\omega_n = 30 \Rightarrow \zeta = 1$$

Q. $C(s) = \frac{400}{s^2 + 90s + 900}$

$$s^2 + 2\zeta\omega_n s + \omega_n^2$$

$\omega_n = 30, \zeta = 1.5$

$$\text{Q. } G(s) = \frac{80}{s(s+18)}$$

for unity feedback

$$1 + G(s)H(s) = 0$$

$$1 + \frac{80}{s(s+18)} = 0$$

$$s^2 + 18s + 80 = 0$$

$$\omega_n = \sqrt{80}$$

$$2\xi\omega_n = 18$$

$$2\xi \times \sqrt{80} = 18$$

$$\rightarrow \frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

$$\text{If } r(t) = U(t) \Rightarrow R(s) = \frac{1}{s}$$

$$C(s) = \frac{\omega_n^2}{s(s^2 + 2\xi\omega_n s + \omega_n^2)}$$

$$C(t) = 1 - \frac{e^{-\xi\omega_n t} \sin(\omega_d t + \phi)}{\sqrt{1-\xi^2}}$$

$$\Rightarrow \omega_d = \omega_n \sqrt{1-\xi^2}$$

$$\Rightarrow \cos\phi = \xi$$

$$\rightarrow \phi = \cos^{-1}\xi$$

Characteristic Eqn. is

$$s^2 + 2\xi\omega_n s + \omega_n^2 = 0$$

$$s = \frac{-2\xi\omega_n \pm \sqrt{4\xi^2\omega_n^2 - 4\omega_n^2}}{2}$$

$$s = \frac{-2\xi\omega_n \pm 2j\sqrt{\omega_n^2 - \xi^2\omega_n^2}}{2}$$

$$s = -\xi\omega_n \pm j\omega_n \sqrt{1-\xi^2} \Rightarrow \boxed{s = -\xi\omega_n \pm j\omega_n \sqrt{1-\xi^2}}$$

$$s = -\alpha \pm j\omega_d \quad (5)$$

where $\alpha = \xi \omega_n \rightarrow$ Damping factor

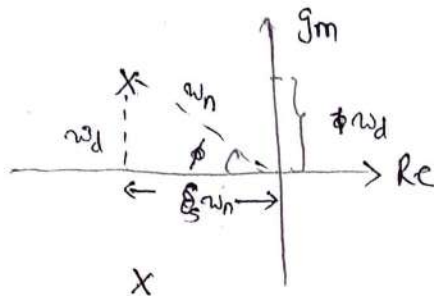
$\xi \rightarrow$ Damping ratio

$$\text{time constant} = \frac{1}{\xi \omega_n}$$

So, Inverse of damping factor is known as time constant.

~~So~~

$$\cos \phi = \xi$$



x

$\alpha > 0, \xi > 0$
1. Positive Damping \rightarrow Stable

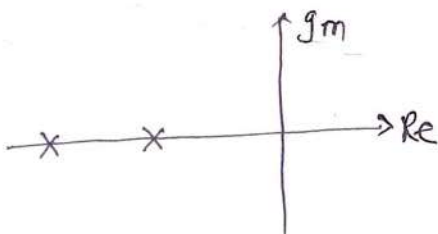
$\alpha < 0, \xi < 0$
2. -ve Damping \rightarrow unstable

$\alpha = 0, \xi = 0$
3. Zero damping. \rightarrow Marginally stable or unstable.

$$\rightarrow \frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

\rightarrow Determination of Step Response for II-order control system

Case I:-

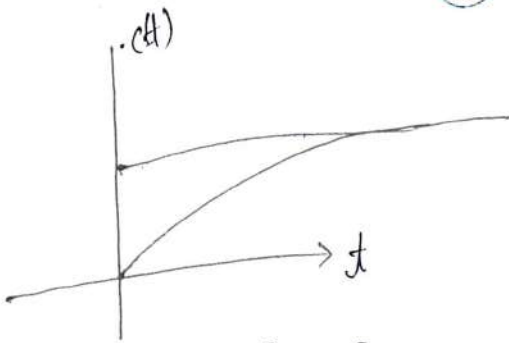


$\xi > 1$

i) under-damped $\xi < 1$

ii) over-damped $\xi > 1$

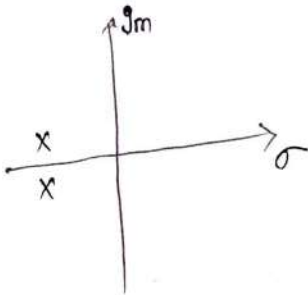
iii) critical " $\xi = 1$



Step Response

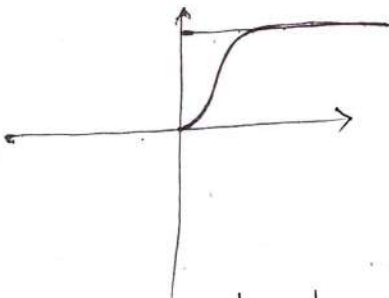
Case 2:- critical Damped

Roots, are repeated in nature

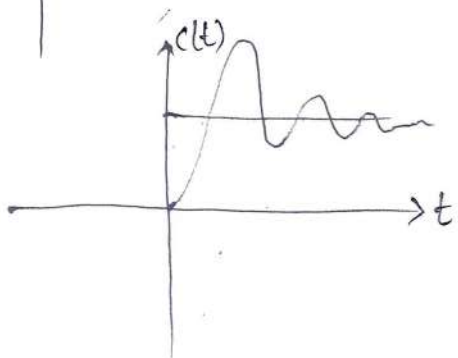
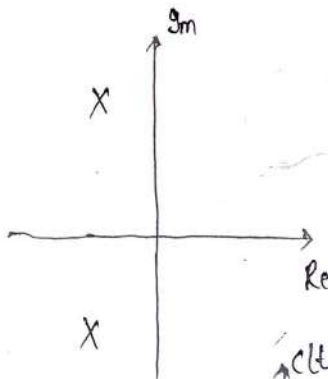


$$\frac{4}{s^2 - 14s + 4}$$

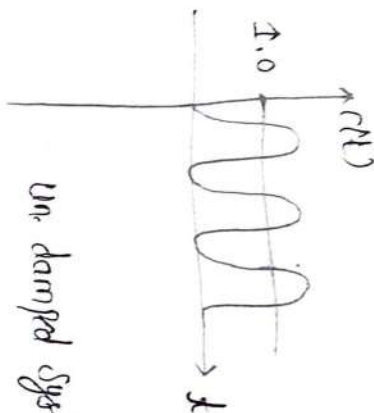
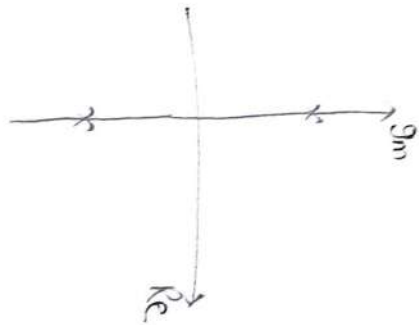
$$(s+2)^2 = 0$$



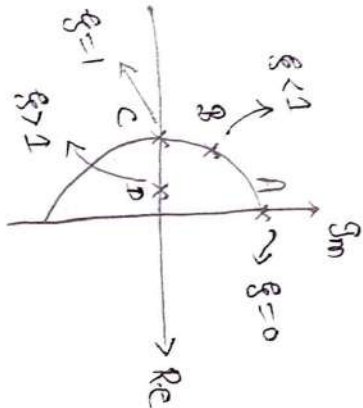
Case 3:- $\xi < 1$ under-damped



$\xi = 0$



for live & zero damping \rightarrow



1. $\xi = 0$ Roots are imaginary \rightarrow un-damped

2. $\xi < 1$ Roots are complex \rightarrow under-damped

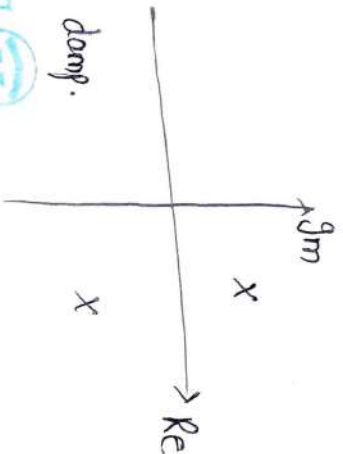
3. $\xi = 1$ \rightarrow Roots are real & complex. Equal \rightarrow critical damped

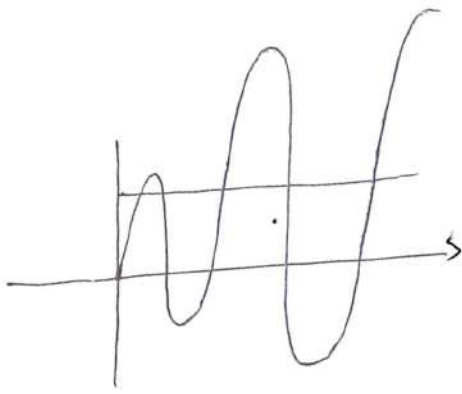
4. $\xi > 1$ \rightarrow Roots are real & unequal \rightarrow over-damped

#-ve Damping \rightarrow

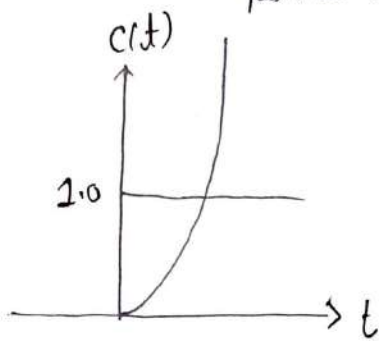
1. $-1 < \xi < 0$

Similarly to under-damped system of live damp.





Case 2:- $\zeta < -1 \rightarrow$ Similar to over-damped system of positive damping.



Q.
IES-05

An under-damped sec. order system ^{with} -ve damping will have two roots

- A. one -ve real axis as real roots
- B. left hand side of complex plane as complex roots.
- C. Right hand side of complex plane as complex conjugates. ()
- D. on the real axis as real roots.

Q.
IES 95

for a II-order system damping ratio ζ is $0 < \zeta < 1$
The roots of charact. polynomial are

- A) Real but not Equal
- B) Real & Equal
- C) complex conjugate ()
- D) Imaginary.

Q.
GATE

The poles of a continuous time oscillations

- A) conjugate ()
- B) Equal
- C) Zero
- D) Not Related

Time - Domain Specification →

- i) Peak - time
- ii) Peak - overshoot
- iii) Rise - time
- iv) Delay Time
- v) Settling time

Peak - time

o/p of unit step inp to a sec.-order system is

$$y(t) = 1 - \frac{e^{-\zeta\omega_n t}}{\sqrt{1-\zeta^2}} \sin(\omega_d t + \phi)$$

Where, $\cos\phi = \zeta$
 $\omega_d = \omega_n \sqrt{1-\zeta^2}$

for peak time $\frac{dy(t)}{dt} = 0$

$$t_p = \frac{n\pi}{\omega_d}$$

∴ Peak - Time

$$t_p = \frac{n\pi}{\omega_n \sqrt{1-\zeta^2}}$$

$n = 1, 2, 3, 4, \dots$ So on.

If $n = 1, 3, 5, 7, \dots$ overshoot

If $n = 2, 4, 6, 8, \dots$ undershoot

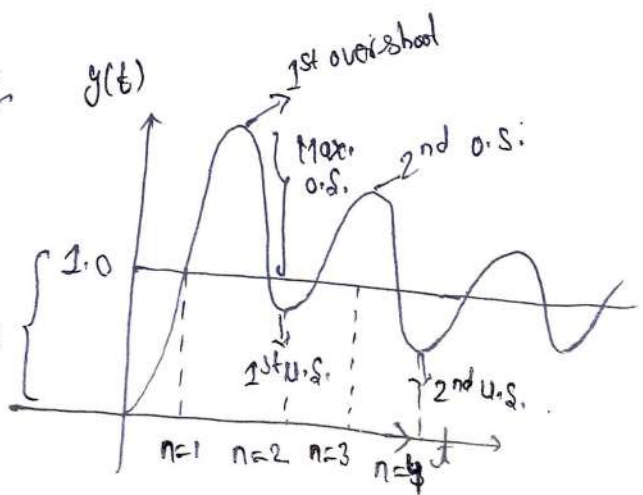
$n = 1 \rightarrow$ 1st overshoot

$n = 2 \rightarrow$ 1st undershoot

$n = 3 \rightarrow$ 2nd overshoot

$n = 4 \rightarrow$ 2nd undershoot

y_{max}
 y_{ss}



Max o.s
 $= y_{max} - y_{ss}$



$$\% \text{ overshoot} = \frac{\text{Max. overshoot}}{y_{ss}}$$

$y_{ss} \rightarrow$ Steady-state

$$\% \text{ overshoot} = \frac{y_{\max} - y_{ss}}{y_{ss}} \times 100 \%$$

$$\% M_p = \frac{e^{-\pi \xi / \sqrt{1 - \xi^2}}}{\text{~~1~~}} \times 100 \%$$

$$\left. \begin{array}{l} y_{\max} = y(t = t_p) \\ y_{ss} = 1 \end{array} \right\} \% M_p = \frac{y(t = t_p) - 1}{1} \times 100 \%$$

for more stability values of peak overshoot must be as larger possible.

what value of K , for a unity feedback system with

$$G(s) = \frac{K}{s(1+s)}$$

peak overshoot of 50%

a) 0.53

b) 5.3 (✓)

c) 0.6

d) 0.047

$$e^{\frac{\pi \xi}{\sqrt{1-\xi^2}}} = 2$$

$$\frac{\pi \xi}{\sqrt{1-\xi^2}} = \ln 2$$

$$\frac{\pi \xi}{\sqrt{1-\xi^2}} = 0.6932$$

$$\log_2 2 = 0.3010$$

$$\log_e 2 = \frac{\log_2 2}{\log_2 e} = 2.303 \log_2 2$$

$$\therefore \frac{\xi}{\sqrt{1-\xi^2}} = \frac{0.6932}{3.14} = 0.22$$

$$\frac{\xi}{\sqrt{1-\xi^2}} = 0.22 \Rightarrow \frac{\xi^2}{1-\xi^2} = 0.484$$

$$\xi^2 = 0.326 \approx 0.33$$

$$\xi \approx 0.57$$

$$G(s) = \frac{K}{s(1+s)}$$

$$1 + G(s)H(s) = 0$$

$$1 + \frac{K}{s(1+s)} = 0$$

$$s(1+s) + K = 0$$

$$s^2 + s + K = 0$$

$$s^2 + 2\xi\omega_n s + \omega_n^2 = 0$$

$$\omega_n^2 = K \Rightarrow \omega_n = \sqrt{K}$$

$$2\xi\omega_n = 1$$

$$2\xi\sqrt{K} = 1$$

$$2 \times 0.57\sqrt{K} = 1$$

$$\sqrt{K} = \frac{1}{2 \times 0.57}$$

$$K = 0.76$$



$$1 + G(s)H(s) = 0$$

(3)

$$1 + \frac{1}{s(s+1)} = 0$$

$$s(s+1) + 1 = 0$$

$$s^2 + s + 1 = 0$$

$$s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$$

$$\omega_n = 1, \quad 2\zeta\omega_n = 1$$

$$2\zeta = 1$$

$$\zeta = 1/2 = 0.5$$

$$M_p = e^{-\pi\zeta/\sqrt{1-\zeta^2}}$$

$$= e^{-\pi \times 1/2 / \sqrt{1-1/4}} = e^{-\frac{\pi/2}{\sqrt{3/4}}} = 0.163$$

$M_p = \text{Max overshoot} - e_{ss}$

Delay-time:- It is time required for response to reach 50% of final value in first attempt.

$$t_d = \frac{1 + 0.7\zeta}{\omega_n} \quad \boxed{0 < \zeta < 1.0}$$

$$\left. \begin{array}{l} t_d \propto \zeta \\ t_d \propto \frac{1}{\omega_n} \end{array} \right\}$$

Rise-time:-

0 - 100% of its final value for under-damped.

10% - 90% of its final value for over-damped.

$$y(t) = 1 - \frac{e^{-\zeta\omega_n t} \sin(\omega_d t + \phi)}{\sqrt{1-\zeta^2}}$$

for calculation of Rise time simply put

$$y(t) = 1$$

$$\sin(\omega_d t + \phi) = 0$$



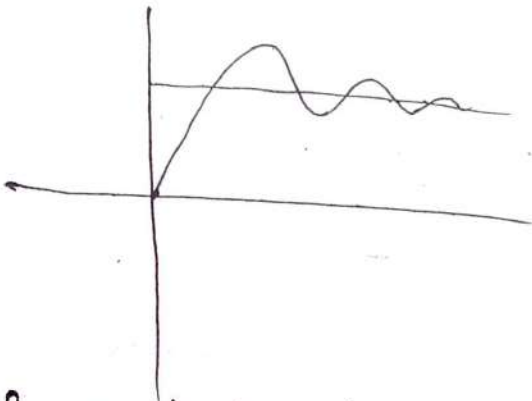
$$\omega_d t_r + \phi = \pi$$

$$t_r = \frac{\pi - \phi}{\omega_d}$$

$$t_r = \frac{0.8 + 2.5\xi}{\omega_n}$$

$$0 < \xi < 1.$$

settling time \rightarrow It is time required by system to reach & stay within a specified tolerance band (2% or 5%) of its final value.



for 2% band it takes 4,
time const.

$$t_s = \frac{4}{\xi \omega_n}$$

for 5% band it
takes 3, time const.

$$t_s = \frac{3}{\xi \omega_n}$$

These formula are not valid for over-damped.

for over-damped

$$t_s = \frac{2\xi}{\omega_n} \text{ for } 2\% \text{ band}$$

The T/F of a system is

$$\frac{b(s)}{a(s)} = \frac{100}{(s+1)(s+100)}$$

to system the approx. settling time for 2% criteria is

a) 100 sec.

c) 1 sec. (+)

b) 4 sec. ✓

d) 0.01 sec.